Inasmuch as  $(u^I, c^I)$  is a  $\sigma$  correction and  $\delta, \sigma \leq 1$ ,  $(u^I, c^I)$  may be expanded in a series in  $\delta$  and evaluated at order  $\delta^0$  to obtain a valid first-order correction. The resultant equations may be integrated to provide

$$\tau^{I} \exp \left[ \int dx / u^{0} \right] + \int \exp \left[ \int (u^{0} + \tau^{0}) dx / u^{0^{2}} \right]$$

$$\times d \left\{ u^{I} \exp \left[ \pm \int \tau^{0} dx / u^{0^{2}} \right] \right\} = \text{const}$$
(34)

$$q^l \exp \left[ \int \! \mathrm{d}x/u^0 \, \right] + \int \! \exp \left[ \int \! \mathrm{d}x/u^0 \, \right] \mathrm{d} \left( c^I + c^0/2 \right)$$

$$= \int q^{0} u^{I} \exp \left[ \int dx / u^{0} \right] dx / u^{0^{2}} + \text{const}$$
 (35)

which, in the far field, provides an additional exponential dependence proportional to  $\sigma$ .

Of primary importance here is the interpretation that the effect of  $\delta$ , in real flows for which  $\sigma > 0$ , is to allow the transported scalar to communicate velocity field details throughout the field more rapidly than the momentum equation. In particular, due to the fact that the u equation is c dependent, information on localized peculiarities in u field data is transmitted to, and affects the solution of, the (remote) u field solution via the transported scalar.

## Acknowledgment

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# Noniterative Cross-Flow Integration for the Pressure-Split Analysis of Subsonic Mixing-Layer Problems

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### Introduction

THE spatial marching analysis of subsonic, quasiparabolic mixing-layer problems is commonly performed using the numerical artifice of pressure splitting. <sup>1-3</sup> In the pressure-split approach, the governing parabolized Navier-Stokes (PNS) equations are spatially integrated with the streamwise pressure gradient "imposed," and the cross-flow pressure variation determined aposteriori, at each integration step, from the coupled solution of the continuity and cross-flow

momentum equations. Thus, the stepwise integration is comprised of 1) a standard parabolic integration yielding the streamwise component of velocity and pertinent scalar (total enthalpy, species) and turbulence model variables; and 2) an elliptic-like cross-flow integration yielding the cross-flow velocity components and pressure variation. Subsequent upgrades to the solution based on a global pressure iteration can be performed in regions with strong pressure gradients.

While the parabolic streamwise integration procedures are comparable in most models, the details of the local and global pressure-splitting procedures are problem dependent, and the cross-flow solution techniques vary widely. The continuity and cross-flow momentum equations are strongly coupled through the pressure, density, and cross-flow velocity derivatives, and are solved in an iterative manner. In the popular approach of Patankar and Spalding, a pressurecorrection equation arrived at from the continuity equation (with cross-flow momentum constraints) is used to determine the cross-flow pressure variation, while the cross-flow velocities are determined from the momentum equations. In contrast, the two-dimensional iterative procedure of Bradshaw and coworkers<sup>3,4</sup> employs the continuity equation to determine the cross-flow velocity and the normal momentum equation to determine the pressure variation.

In utilizing pressure-split methodology for the twodimensional analysis of curved wall jets<sup>5</sup> and subsonic regions of underexpanded free jets<sup>6,7</sup> (i.e., behind Mach disks and between the jet mixing-layer sonic line and jet outer edge), Dash and coworkers initially utilized the cross-flow procedure of Bradshaw and coworkers. In assessing this procedure, it was found that the iterative sweeps required between the continuity and cross-flow momentum equation solutions could be eliminated by combining these equations into a unified equation for the cross-flow velocity. This provides a considerable savings in overall computer time and eliminates possible convergence problems occurring in iterative approaches. This Note describes this new noniterative procedure and its application to a simple two-dimensional curved wall jet problem.

It should be noted that this new technique extends the efficiency of single-sweep pressure-splitting methodology to a level whereby PNS mixing solutions can be obtained in slightly more time than that required for standard parabolic mixing solutions. The results provided using this rapid procedure are identical to those obtained using previous iterative, pressure-split methodology as ascertained by numerical experiments. Hence, the favorable comparisons with data obtained using the curved wall jet and free jet models of Refs. 5-7 remain unchanged, and, the results of Bradshaw and coworkers<sup>3,4</sup> would be reproduced using this approach if all other aspects of the computational procedure and turbulence modeling were duplicated.

#### **Cross-Flow Analysis**

The overall mixing-layer analysis is performed in mapped, surface-oriented curvilinear coordinates using the upwind, implicit formulation described in Ref. 5. With the rectangular mapping

$$\xi = s \qquad \eta = n/\delta(s) \tag{1}$$

(where s is the streamwise direction, n is normal to it, and  $\delta(s)$  is the width of the mixing zone) the parabolized, planar normal momentum equation can be written<sup>5</sup>:

$$\rho U \frac{\partial V}{\partial \xi} + \rho \tilde{V} \frac{\partial V}{\partial \eta} + bh \frac{\partial P}{\partial \eta} + K\rho U^2 = g_v$$
 (2)

where  $\tilde{V} = bhV - aU$ ; K is the curvature, h a curvature parameter (=1-nK); a and b mapping parameters; and  $g_v$  contains the laminar and turbulent stress terms.<sup>5</sup>

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The mapped continuity equation takes the form

$$\frac{\partial}{\partial \xi} \left( \frac{\rho U}{b} \right) + \frac{\partial}{\partial \eta} \left( h \rho V - \frac{a}{b} \rho U \right) = 0 \tag{3}$$

The density derivatives in Eq. (3) can be related to pressure, velocity, and total enthalpy derivatives via differentiating the state relation yielding

$$d\rho = [\gamma dP + (\gamma - I)\rho (UdU + VdV - dH)]/C^2$$
 (4)

where  $C^2 = \gamma P/\rho$ . Noting that

$$\frac{\partial P}{\partial \xi} = \frac{\partial P^*}{\partial s} + a \frac{\partial P}{\partial \eta}$$

where  $\partial P^*/\partial s$  is the imposed streamwise pressure gradient, combining Eqs. (3) and (4) yields the modified continuity relation

$$\gamma \left( a + \frac{\tilde{V}}{U} \right) \frac{\partial P}{\partial \eta} + (\gamma - I) \rho V \frac{\partial V}{\partial \xi}$$

$$+ \left[ \frac{\rho b h C^2}{U} + (\gamma - I) \frac{\rho V \tilde{V}}{U} \right] \frac{\partial V}{\partial \eta} = g_{\rho}$$
(5)

where the source term  $g_p$  (whose terms are known from the streamwise parabolic integration performed apriori) is given by

$$\begin{split} g_{p} &= \rho b C^{2} \frac{\partial}{\partial \eta} \left( \frac{a}{b} \right) + K \rho C^{2} \frac{V}{U} - \left[ \frac{\rho C^{2}}{U} + (\gamma - I) \rho U \right] \frac{\partial U}{\partial \xi} \\ &+ \left[ \frac{\rho a C^{2}}{U} - (\gamma - I) \rho \tilde{V} \right] \frac{\partial U}{\partial \eta} + (\gamma - I) \rho \left[ \frac{\partial H}{\partial \xi} + \frac{\tilde{V}}{U} \frac{\partial H}{\partial \eta} \right] \\ &- \gamma \frac{\partial P^{*}}{\partial s} + \frac{\rho C^{2}}{b} \frac{\partial b}{\partial \xi} \end{split}$$

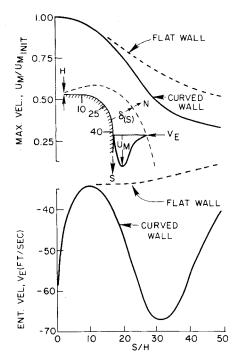


Fig. 1 Streamwise variations of maximum jet velocity and entrainment velocity for wall jet on flat and curved surfaces.

Equations (2) and (5) uniquely define the cross-flow velocity and pressure variation and are readily combined into a single equation for V by eliminating the  $\partial P/\partial \eta$  term. This yields an equation of the form

$$A_{1} \frac{\partial V}{\partial \xi} + A_{2} \frac{\partial V}{\partial \eta} = G(V)$$
 (6)

which can be solved without requiring any assumptions concerning the cross-flow pressure variation. Equation (6) has been integrated explicitly (treating the stress terms as forcing functions) via a two point upward integration from the wall or axis of symmetry. Utilizing this approach within the confines of a fully implicit procedure for the solution of the streamwise parabolic equations<sup>5</sup> has presented no stability problems. The cross-flow pressure variation is subsequently determined via a downward trapezoidal integration of Eq. (2) utilizing the predicted V distribution and initiated with the prescribed outer-edge pressure.

#### **Curved Wall Jet Calculation**

The calculation described was performed for an initially sonic wall jet exhausting into still air, undergoing a 90 deg turn over the convex surface depicted in Fig. 1. The surface curvature K is zero for the first 10 slot heights, varies quadratically from 10 < S/H < 40 with a peak value at S/H=25 corresponding to a radius of curvature of 12.5 slot heights, and is zero for S/H < 40. The predicted jet boundary growth (Fig. 1) exhibits that the jet width at the position of maximum curvature is comparable to the surface radius of curvature. The predicted decay of maximum streamwise velocity and the variation of the entrainment (outer, normal influx) velocity are compared with the corresponding values for a flat surface in Fig. 1. The curvature significantly enhances the rate of mixing and, hence, the streamwise velocity decay and inward entrainment velocities. Note the rapid departure of the curved and flat wall entrainment velocity distributions beyond S/H = 10; the location of peak entrainment velocity at  $S/H\sim30$ , downstream of the peak curvature position; and the rapid approach of the curved wall value to the flat wall value beyond S/H=40, where the curvature terminates. The calculations were performed using the curvature-corrected<sup>8</sup> two-layer turbulence model (inner damped Van Driest, outer  $k\epsilon$ ) described in Ref. 5.

The predicted pressure variations along the curved surface utilizing the viscous and inviscid forms of the normal momentum equation are exhibited in Fig. 2. The

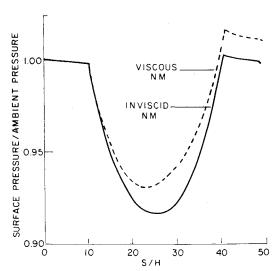


Fig. 2 Streamwise variation of surface pressure using viscous and inviscid forms of normal momentum equation.

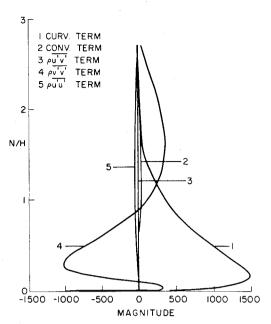


Fig. 3 Contribution of curvature, convective, and turbulent stress terms to normal pressure variation across wall jet at S/H = 15.

viscous/turbulence terms tend to reduce the deviation of the wall pressure from the external stream value. Note that the inviscid pressure is minimum at the peak curvature position of S/H = 25 while the viscous value is minimum somewhat upstream. The contribution of the various terms in the normal momentum equation to the pressure variation across the curved wall jet is exhibited in Fig. 3 at the position S/H=15. The dominant term, as expected, is the surface curvature term  $(K\rho U^2)$ . Using standard Boussinesq modeling for the turbulent stress terms, viz.

$$-\rho \overline{u_i'u_j'} = -\frac{2}{3}\rho k \delta_{ij} + \mu_t \left[ \left( \frac{\partial U_i}{\partial X_j} + \frac{\partial U_j}{\partial X_i} \right) - \frac{2}{3}\operatorname{div} \bar{V} \right]$$
 (7)

(see Ref. 5 for the specific parabolized terms retained in curvilinear s, n coordinates) the dominant viscous contribution comes from the  $\rho \overline{v'v'}$  stress term. This term acts counter to the surface curvature term over a significant portion of the wall jet. Comparable profiles at other stations and further details of this calculation are available in Ref. 9.

# **Concluding Remarks**

The noniterative cross-flow procedure described has proven to be an efficient and reliable improvement to the twodimensional iterative cross-flow procedure of Bradshaw and coworkers.<sup>3,4</sup> Comparable results with the Bradshaw method typically required two to three iterative sweeps at each station. Parabolic mixing-layer codes can be extended readily to incorporate this cross-flow procedure, and their "direct coupling" with an external potential flow solution<sup>3,4</sup> provides a simple extension of boundary-layer interactive concepts to situations with large normal pressure variations. The authors have formulated a "semi-iterative" extension of this approach to three-dimensional mixing-layer problems via analogous manipulations of the continuity and cross-flow momentum equations which will be described in a future article. 10

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# **Emmons Spot Forcing** for Turbulent Drag Reduction

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# Nomenclature

|                  | 11011101101101                                                                               |
|------------------|----------------------------------------------------------------------------------------------|
| $C_F/C_{F_{HF}}$ | = skin-friction coefficient normalized by skin-<br>friction coefficient at maximum frequency |
| f                | = frequency, Hz                                                                              |
| $R_{	heta}$      | = Reynolds number based on momentum thickness                                                |
| St               | = Strouhal number, = $f \delta / U \infty$                                                   |
| $U_{\infty}$     | = freestream velocity                                                                        |
| $V_w'$           | = acoustic perturbation velocity                                                             |
| $\boldsymbol{Z}$ | = transverse coordinate                                                                      |
| δ                | = boundary-layer thickness                                                                   |
| $\theta$         | = boundary-layer momentum thickness                                                          |
|                  |                                                                                              |

### Introduction

THE viscous or skin-friction drag associated with a turbulent boundary layer accounts for approximately 50% of the drag on conventional takeoff and landing (CTOL) aircraft and surface ships and nearly all of the pumping power in long-distance pipelines. Reduction of this viscous drag would allow longer range, reduced fuel volume/cost, and/or

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